



# Prograde and meandering wall modes in rotating Rayleigh–Bénard convection with conducting walls

S. Ravichandran<sup>1,2,†</sup> and J.S. Wettlaufer<sup>1,3</sup>

<sup>1</sup>Nordita, KTH Royal Institute of Technology and Stockholm University, Stockholm 10691, Sweden

<sup>2</sup>Centre for Climate Studies, Indian Institute of Technology Bombay, Mumbai 400076, India

<sup>3</sup>Yale University, New Haven, Connecticut 06520-8109, USA

(Received 6 April 2023; revised 5 September 2024; accepted 8 September 2024)

We use direct numerical simulations to study convection in rotating Rayleigh–Bénard convection in horizontally confined geometries of a given aspect ratio, with the walls held at fixed temperatures. We show that this arrangement is unconditionally unstable to flow that takes the form of wall-adjacent convection rolls. For wall temperatures close to the temperatures of the upper or lower boundaries, we show that the base state undergoes a Hopf bifurcation to a state comprised of spatiotemporal oscillations – ‘wall modes’ – precessing in a retrograde direction. We study the saturated nonlinear state of these modes, and show that the velocity boundary conditions at the upper and lower boundaries are crucial to the formation and propagation of the wall modes: asymmetric velocity boundary conditions at the upper and lower boundaries can lead to prograde wall modes, while stress-free boundary conditions at both walls can lead to wall modes that have no preferred direction of propagation.

**Key words:** rotating flows, bifurcation

## 1. Introduction

Rotating convection is an essential process in, among many other systems, stars (Schumacher & Sreenivasan 2020), protoplanetary disks (Gerbig & Laughlin 2022), planetary interiors (Aurnou *et al.* 2015; Bercovici 2015; Aurnou, Horn & Julien 2020), Earth’s atmosphere and oceans (e.g. Emanuel 1994; Marshall & Schott 1999; Gayen & Griffiths 2022), and the cryosphere (e.g. Wells & Wettlaufer 2008; Ravichandran

† Email address for correspondence: [sravichandran@iitb.ac.in](mailto:sravichandran@iitb.ac.in)

& Wettlaufer 2021; Cenedese & Straneo 2023). Moreover, rotating Rayleigh–Bénard convection acts as a model system for these phenomena, and the study of pattern formation, such as transient axisymmetric rings observed during spin-up (e.g. Boubnov & Golitsyn 1986; Vorobieff & Ecke 1998; Zhong, Patterson & Wettlaufer 2010; Ravichandran & Wettlaufer 2020) and travelling waves observed near the onset of convection (Ecke, Zhong & Knobloch 1992; Ning & Ecke 1993; Liu & Ecke 1997, 1999). In typical model studies of rotating Rayleigh–Bénard convection, both theory and numerical simulations employ horizontally periodic domains, whereas experiments do not, leading in part to disagreements. For instance, Rossby (1969) observed that in rotating Rayleigh–Bénard convection with the typical no-slip experimental boundary conditions, the onset of convection occurs for significantly smaller thermal forcing than predicted by the linear stability analysis of Chandrasekhar (1953). Although explanations of this apparent disagreement between theory and experiment can be traced to the basic differences between boundary conditions (BCs) and finite amplitude perturbations (e.g. Veronis 1966, 1968; Herrmann & Busse 1993; Kuo & Cross 1993), there are many interesting outstanding questions (see Ecke 2023). For example, measurements suggest that the peripheral modes may also be responsible for the mismatch between the Nusselt numbers in laboratory experiments and numerical simulations (Kunnen, Clercx & Heijst 2013; Wit *et al.* 2020; Ecke, Zhang & Shishkina 2022).

In the absence of rotation, the onset of convection in a horizontally unbounded layer of fluid of depth  $H$  across which a temperature difference  $\Delta T$  is maintained, is governed by the dimensionless Rayleigh number

$$Ra = \frac{g\alpha\Delta TH^3}{\nu\kappa}, \quad (1.1)$$

where  $g$  is the acceleration due to gravity and  $\nu$  and  $\kappa$  are the viscosity and thermal diffusivity of the fluid. The onset of convection occurs when a critical Rayleigh number,  $Ra_c^{bulk}$ , is exceeded, where  $Ra_c^{bulk} = O(10^3)$ , with the exact value depending on the BCs. Chandrasekhar (1953) showed that, independent of the Prandtl number  $Pr = \nu/\kappa$ , the instability that leads to convection is non-oscillatory.

Rotation about the vertical axis suppresses the effects of buoyancy and thus enhances stability. Therefore, the critical Rayleigh number increases with the rotation rate,  $\Omega$ , as

$$Ra_c^{bulk} \simeq E^{-4/3}, \quad (1.2)$$

where  $E = \nu/2\Omega H^2$  is the Ekman number (Chandrasekhar 1953; Veronis 1966). In contrast to non-rotating systems, the onset of convection in horizontally unbounded rotating Rayleigh–Bénard convection can be oscillatory if the Prandtl number  $Pr < 0.69$  (Chandrasekhar 1953).

Rossby (1969) showed experimentally that convection sets in for much smaller Rayleigh numbers than predicted by (1.2), the possible origins of which were discussed contemporaneously by Veronis (1968) as being associated with BCs. Of relevance to our study, the experiments by Ecke *et al.* (1992), supported by linear stability analysis by Goldstein *et al.* (1993), showed that the wall-adjacent convection takes the form of a travelling wave with a phase speed opposite to the sense of rotation. They also showed that the system undergoes a Hopf bifurcation at a critical Rayleigh number, following which the travelling wave appears.

Herrmann & Busse (1993) and Kuo & Cross (1993) independently showed that, in the asymptotic limit of  $E \rightarrow 0$ , the critical Rayleigh number for the onset of the travelling

waves in confined rotating Rayleigh–Bénard convection with adiabatic walls is

$$Ra_{cw}^{ZF} = \pi^2(6\sqrt{3})^{1/2}E^{-1} = 31.82E^{-1}, \quad (1.3)$$

where the superscript *ZF* denotes ‘zero flux’. Herrmann & Busse (1993) also showed that conducting walls stabilize the wall modes, with a critical Rayleigh number that has the same leading order scaling as that for an infinite layer, but with a smaller prefactor,

$$Ra_{cw}^{FT} = 0.9086(\pi E^{-1})^{4/3} + 2.124(\pi E^{-1})^{7/6} = 4.18E^{-4/3} + 8.08E^{-7/6}, \quad (1.4)$$

where the superscript *FT* denotes ‘fixed temperature’ highlighting the fact that mathematical well-posedness requires the temperature to be prescribed at a conducting boundary. In their analysis, Herrmann & Busse (1993) assume a wall temperature equal to the purely conductive (linear) profile

$$\bar{T}_{wall} = 1 - z. \quad (1.5)$$

The flow, for  $Ra > Ra_{cw}^{FT}$ , again takes the form of travelling waves that propagate in a retrograde sense, against the sense of rotation.

Instead of the linear temperature boundary condition of Herrmann & Busse (1993), if a constant temperature is imposed across the entire height of the walls, the system is unconditionally unstable with the flow taking the form of wall-adjacent convective rolls. Similar rolls were observed by Ning *et al.* (1993) for weak thermal forcing at the walls. Constant wall temperatures may be relevant in the melting of ocean-terminating glaciers due to the rotation-influenced convection, wherein the coupling between convective structures and melting morphology could become important (Ravichandran & Wettlaufer 2021).

For supercritical Rayleigh numbers, the wall modes attain a nonlinear saturated steady state consisting of wall-adjacent regions of upwelling and downwelling flow propagating in a retrograde direction. This nonlinear state has been observed by Lopez *et al.* (2007), Favier & Knobloch (2020), Zhang *et al.* (2020) and Wit *et al.* (2020); and Favier & Knobloch (2020) showed it to be robust to severe non-axisymmetric modifications of the geometry. For Rayleigh numbers well beyond the onset of bulk convection, the nonlinear state becomes the so-called boundary zonal flow (see e.g. Wit *et al.* 2020; Zhang *et al.* 2020; Ecke *et al.* 2022; Wedi *et al.* 2022), which has been shown to be responsible for significant amounts of heat transfer in rotating Rayleigh–Bénard convection (Wit *et al.* 2020).

Motivated by these findings of the influence of the BCs on the dynamical state of wall modes, here we examine the effects of changing the uniform temperature at which the walls and upper and lower boundaries are held, along with the velocity BCs, as summarized in table 1. We show that the steady roll state undergoes a Hopf bifurcation as a function of the wall temperature, leading to wall modes. The wall temperature at which this onset occurs must be found from linear stability analysis, with the steady rolls as the base state. We study the nonlinear state of the resulting instability, comparing it with the nonlinear state of wall modes with adiabatic walls. We find that the velocity BCs at the upper and lower boundaries control the direction of propagation of the wall modes, and prograde wall modes can arise for suitable velocity BCs. Finally, by studying both cuboidal and cylindrical geometries, we confirm the findings of Favier & Knobloch (2020) that the nonlinear wall mode state is robust to non-axisymmetric geometric modifications.

The rest of the paper is organized as follows. In § 2, we describe the geometry of the problem and the numerical method used for the simulations. In § 3, we present results from the numerical simulations as the wall temperature, the velocity BC at the upper boundary,

BC	$z = 0$	$z = 1$	Walls
SYMNS	$u = v = w = 0, \theta = 1$	$u = v = w = 0, \theta = 0$	$u = v = w = 0, \theta = \theta_w$ or $\partial\theta/\partial n = 0$
ASYM	$u = v = w = 0, \theta = 1$	$w = 0, \partial(u, v)/\partial z = 0, \theta = 0$	$u = v = w = 0, \theta = \theta_w$ or $\partial\theta/\partial n = 0$

Table 1. The combinations of BCs considered. Symmetric (SYMNS) BCs have no-slip upper and lower boundaries, while asymmetric (ASYM) BCs have stress-free upper and no-slip lower boundaries. Consequences of symmetric stress-free (SYMFS) BCs, defined in analogy with SYMNS BCs, and of stress-free BCs at all boundaries (ALLFS) are examined in § 3.2.

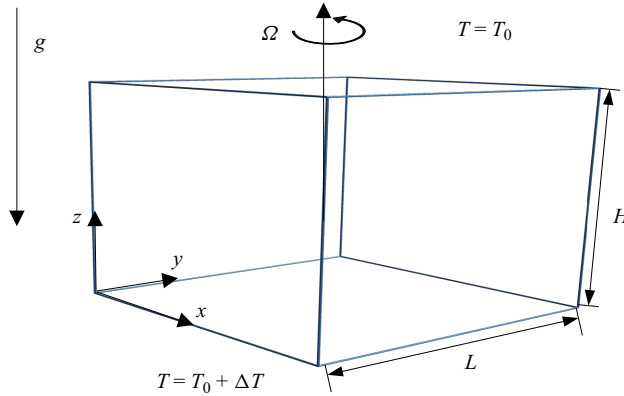


Figure 1. Our simulations are performed in a cuboidal volume of square cross-section and aspect ratio  $L/H = 2$ , so that the domain is the region  $|x| \leq 1, |y| \leq 1, 0 \leq z \leq 1$ . The walls are either adiabatic or held at a constant temperature. The container rotates about the vertical axis with a constant angular velocity, with gravity pointing vertically down.

the horizontal cross-section of the domain and the Prandtl number are varied, compare our results with other known travelling wave solutions in rotating Rayleigh–Bénard convection, and discuss the effects of asymmetric velocity BCs at the upper and lower boundaries on the direction of propagation of the wall-adjacent spatiotemporal patterns. We conclude in § 4.

## 2. Set-up and numerical simulations

The domain is a rectangular volume of height  $H$  and width  $L$ , with an aspect ratio  $\Gamma = L/H = 2$ , shown schematically in figure 1. The system rotates about the vertical  $z$  axis with a constant angular velocity  $\Omega$ .

We make the Boussinesq approximation, so that fluid properties are assumed to be constant and the flow is assumed to be incompressible. We non-dimensionalize the governing equations using the length scale  $H$  and the buoyancy velocity scale  $U_b = (g\alpha\Delta TH)^{1/2}$ . The non-dimensional governing equations become

$$\frac{D\mathbf{u}}{Dt} = -\nabla p - \frac{\sqrt{Pr}}{E\sqrt{Ra}}\mathbf{e}_z \times \mathbf{u} + \left(\frac{Pr}{Ra}\right)^{1/2} \nabla^2 \mathbf{u} + \mathbf{e}_z \theta, \tag{2.1}$$

$$\nabla \cdot \mathbf{u} = 0, \tag{2.2}$$

$$\frac{D\theta}{Dt} = \left(\frac{1}{RaPr}\right)^{1/2} \nabla^2 \theta. \tag{2.3}$$

We consider the combinations of BCs listed in [table 1](#). The temperature of the lower boundary is fixed at  $\theta(z = 0) = 1$ , the upper boundary at  $\theta(z = 1) = 0$  and the walls are either adiabatic ( $\partial\theta/\partial n = 0$ ) or have a fixed temperature  $\theta = \theta_w$ , with  $0 \leq \theta_w \leq 1$ . The velocity obeys either no-slip or stress-free BCs on the walls at  $x = \pm 1, y = \pm 1$ , and the lower and upper boundaries at  $z = 0$  and  $z = 1$ , respectively.

Subject to these BCs, (2.1)–(2.3) are solved using the finite volume solver Megha-5, used in previous studies of convection (Ravichandran & Wettlaufer 2020, 2021; Ravichandran, Toppaladoddi & Wettlaufer 2022). The solver uses second-order central differences in space and a second-order Adams–Bashforth time stepping scheme. Simulations are initialized with broadband noise added to the initial conditions which trigger convection. We use a grid resolution of up to  $256^3$  uniformly spaced points in the three space directions and a time step  $dt \geq 1.25 \times 10^{-3}$ . The Nusselt number changes by only a few per cent, with no change in the convection pattern, when the vertical resolution is changed from 128 to 256 grid points; and by approximately 0.1 % when the horizontal resolution is changed from 256 to 512 grid points.

### 3. Results and discussion

The governing dimensionless parameters of (2.1)–(2.3) are the Ekman ( $E$ ), Rayleigh ( $Ra$ ) and Prandtl ( $Pr$ ), numbers as defined above. Simulations are run for a given aspect ratio  $\Gamma$ , and set of BCs. We minimize the amount of computation required by exploring the effects of varying the parameters one at a time around the point;  $E = 10^{-4}, Ra = 10^6, Pr = 1$ , with  $\Gamma = 2$ , and  $0 \leq \theta_w \leq 1$ . For this combination of  $E$  and  $Pr$  in a horizontally unbounded geometry,  $Ra_c^{bulk} \approx 1.5 \times 10^6$ , whereas with insulating walls wall modes appear for  $Ra > Ra_{cw}^{ZF} = 3.2 \times 10^5$ . For walls with the linear temperature profile (see (1.5)),  $Ra_{cw}^{FT} = 1.3 \times 10^6$  and for  $Ra > Ra_{cw}^{FT}$  retrograde propagating wall modes are obtained regardless of the asymmetry in the velocity BCs. This is discussed further in § 3.4.

The initial and BCs described in § 2 lead to the onset of convection everywhere in the domain. The associated bulk flow structure decays away, leaving only the wall attached convection, from which the wall modes emerge and grow into their nonlinear state. The process from which the wall modes emerge from the wall attached convective state involves an instability of a transient base state, and the simulations show a robust and rapid growth into the nonlinear state. Thus, we focus on the latter situation and leave the stability analysis of the secular base state for a standalone study.

#### 3.1. Symmetric and asymmetric velocity BCs

We first examine the flow structures that arise with SYMNS BCs (see [table 1](#)) for  $E = 10^{-4}, Ra = 10^6, Pr = 1, \Gamma = 2$ , while varying  $\theta_w$ . [Figure 2](#) shows that two types of flow may arise; a wall temperature of  $\theta_w = 0.25$  leads to steady wall-attached rolls in the shape of the container, and  $\theta_w = 0.1$  generates retrograde wall modes. The former are similar to the time-averaged flow seen in the experiments of Ning *et al.* (1993) at supercritical Rayleigh numbers, with weak thermal forcing at the imperfectly conducting walls. The retrograde propagation of the wall modes is apparent from the space–time Hövmöller diagram in [figure 3\(a\)](#), where we show the near-wall temperature along a horizontal line. Due to the symmetry of the problem, retrograde wall modes with a similar structure are observed for  $1 - \theta_w = 0.1$ , as seen in [figure 3\(b\)](#). Note that the downward

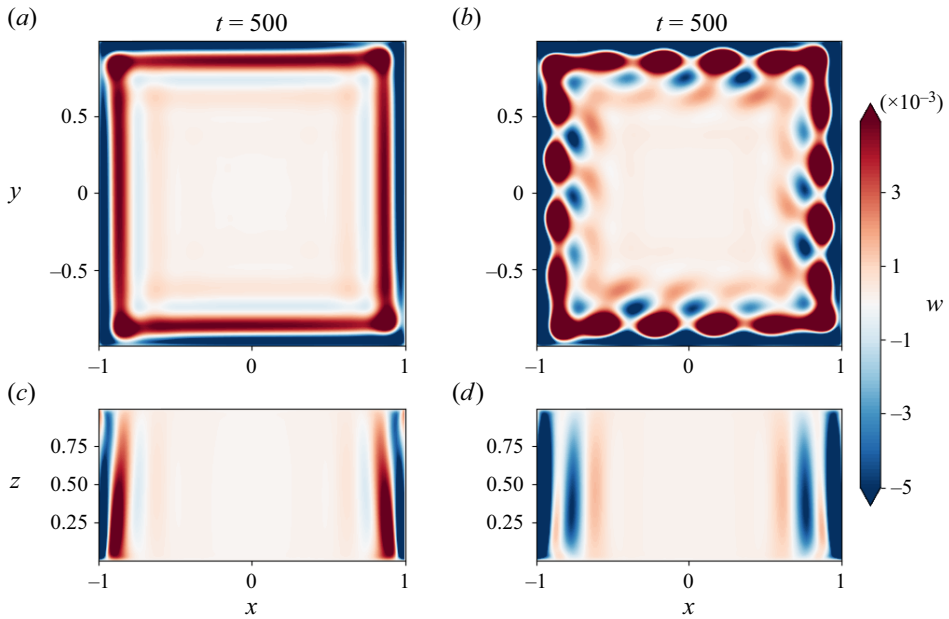


Figure 2. With  $E = 10^{-4}$ ,  $Ra = 10^6$ ,  $Pr = 1$  and SYMNS BCs, we see a steady convective state with wall-adjacent rolls for (a,c)  $\theta_w = 0.25$ ; and wall modes for (b,d)  $\theta_w = 0.1$ . The panels show the (a,b) horizontal and (c,d) vertical cross-sections of the vertical velocity  $w$ .

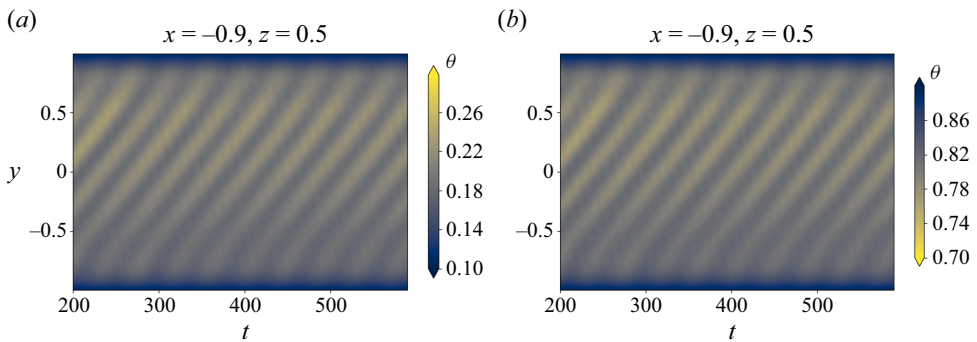


Figure 3. Space–time Hövmöller diagrams of the temperature  $\theta$  for  $E = 10^{-4}$ ,  $Ra = 10^6$ ,  $Pr = 1$ ; and (a)  $\theta_w = 0.1$  and (b)  $\theta_w = 0.9$ . The slope of the patterns shows that the wall modes propagate in a retrograde direction. The symmetry of the Boussinesq equations ensures that the flow patterns for  $\theta_w$  and  $1 - \theta_w$  are similar. Note that the colour scheme is inverted in (b).

vertical velocity seen in the wall-adjacent region for  $\theta_w < 0.5$  (figure 2) would be upward were  $\theta_w > 0.5$ .

Wall modes are also seen with ASYM BCs for wall temperatures close to upper or lower boundary values, as shown in figure 4. In figure 5, we plot the Hövmöller diagrams for the space–time evolution of the wall modes, showing a reversal of the direction of propagation as follows. For small  $\theta_w = 0.05$ , we find retrograde wall modes, as shown in figure 5(a), whereas for large  $\theta_w = 1$ , the wall modes travel in the prograde direction, as shown in figure 5(b). Finally, for intermediate  $\theta_w$  we find convection in steady rolls, as was the case with SYMNS BCs.



Wall modes in rotating Rayleigh–Bénard convection

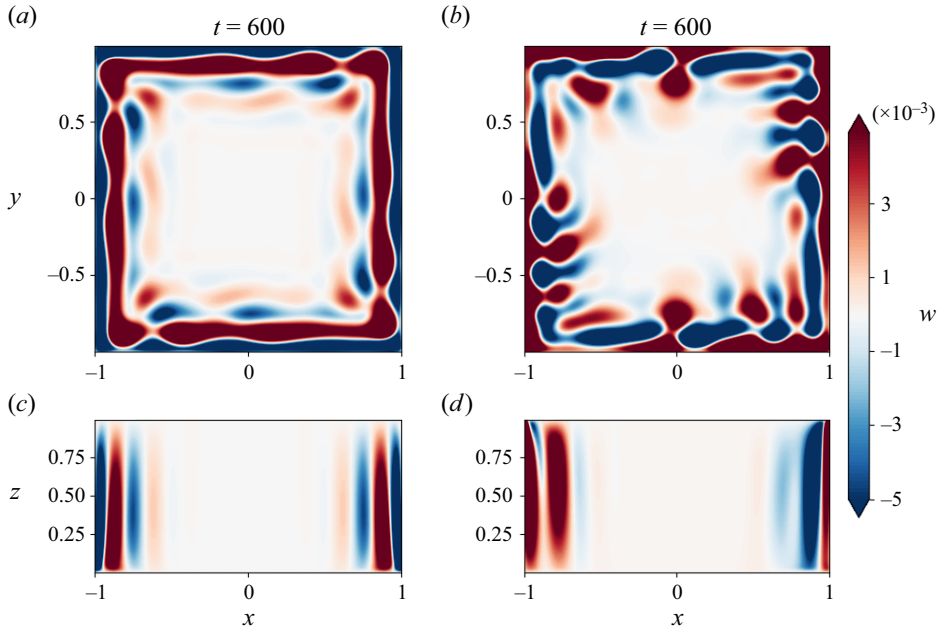


Figure 4. The wall modes observed for  $E = 10^{-4}$ ,  $Pr = 1$ ,  $Ra = 10^6$  and ASYM BCs for (a,c)  $\theta_w = 0.05$  and (b,d)  $\theta_w = 1$ . The panels show the (a,b) horizontal and (c,d) vertical cross-sections of the vertical velocity  $w$ . In (a,c), the wall modes propagate in the retrograde direction, while in (b,d) they propagate in the prograde direction (see figure 5).

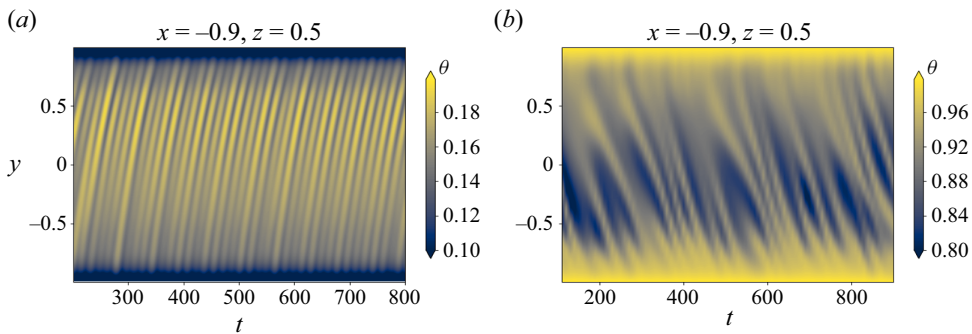


Figure 5. Hövmöller diagrams of the temperature  $\theta$  for the same parameters as in figure 4, showing that for (a)  $\theta_w = 0.05$  the wall modes travel in the retrograde direction, while for (b)  $\theta_w = 1$  they travel in the prograde direction.

Horn & Schmid (2017) find flow features that propagate in the prograde direction for small aspect ratios ( $\Gamma = 0.5$ ) with precession frequencies that are comparable to retrograde modes. However, these flow features are ‘bulk modes’, and not restricted to the near-wall region. Horn & Schmid (2017) associate their bulk modes with the slow modes of Goldstein *et al.* (1993), with precession frequencies much smaller than those of the retrograde modes. Neither of these studies considers the effect of asymmetric velocity BCs, although Goldstein *et al.* (1993) do consider conducting walls. Here, we see prograde wall modes with asymmetric velocity BCs, but only for  $\theta_w \rightarrow 1$ , with precession frequencies that are comparable to the usual retrograde modes. In contrast, for insulating walls, the wall modes are retrograde even for asymmetric BCs; and if the wall temperature is very different from that of the no-slip boundary, say  $\theta_w \lesssim 0.9$ , the flow is comprised of

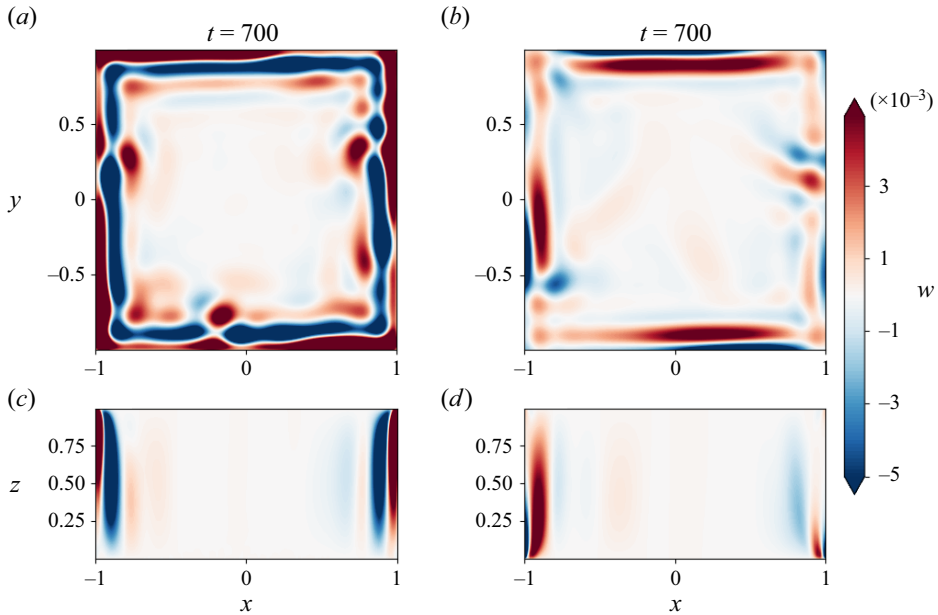


Figure 6. The wall modes observed for  $E = 10^{-4}$ ,  $Ra = 10^6$ ,  $Pr = 1$  and (a,c) SYMFS BCs with  $\theta_w = 0.95$  and (b,d) ALLFS BCs with  $\theta_w = 0.1$ . The panels show the (a,b) horizontal and (c,d) vertical cross-sections of the vertical velocity  $w$ . The wall modes have no fixed direction of propagation (see also figure 7) and the strong four-fold symmetry seen in figures 2 and 4 is lost.

nested rolls. In § 3.4, we explain this behaviour by examining the tangential velocity at the walls.

### 3.2. Additional combinations of BCs

If both the upper and lower boundaries are stress free, and the walls are conducting and obey the no-slip condition (SYMFS BCs, see table 1), wall modes form but they have no clearly discernible direction of propagation. Insulating walls lead to the standard retrograde wall modes, in agreement with earlier studies. Similarly, if both the upper and lower boundaries as well as the conducting walls are stress free (ALLFS BCs), the wall modes that appear have no fixed direction of propagation. Representative snapshots of the wall modes that result from SYMFS and ALLFS BCs are shown in figures 6(a) and 6(b), respectively, with the corresponding Hövmöller diagrams shown in figures 7(a) and 7(b), respectively. A comparison of these figures with the equivalent figures for SYMNS (figures 2 and 3) and ASYM BCs (figures 4 and 5) suggests that while wall modes are observed with conducting walls for suitable wall temperatures  $\theta_w$ , no-slip velocity BCs on at least the upper or lower boundaries are necessary for wall modes to propagate in a definite direction. We explain this behaviour by examining the tangential velocity at the walls in § 3.4. Finally, we note that the flow in figure 6 has lost the strong four-fold symmetric structure shown in figures 2 and 4 for SYMNS and ASYM BCs, respectively.

### 3.3. Cylindrical geometry

The robustness of wall modes to changes in geometry, observed in the experiments of Ning & Ecke (1993) (but not published; see Ecke (2023)), was studied numerically by



## Wall modes in rotating Rayleigh–Bénard convection

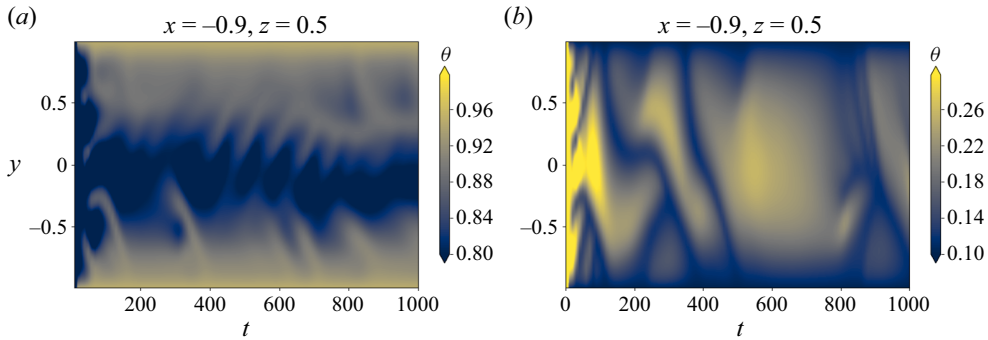


Figure 7. Hövmöller diagrams of the temperature  $\theta$  for the same parameters as in figure 6, showing that for both (a)  $\theta_w = 0.95$  and SYMFS BCs, and (b)  $\theta_w = 0.1$  and ALLFS BCs, the wall modes show no fixed direction of propagation.

Favier & Knobloch (2020). Terrien, Favier & Knobloch (2023) observed that wall modes can be suppressed by obstacles (‘fins’) on the boundary by increasing the thermal forcing required for wall mode onset. Here we show the consistency of the flow structures between cuboidal and cylindrical geometries. We also find that the sharp corners in the cuboidal geometry act to dampen the travelling wall modes.

We show the geometry-independence of our findings by performing simulations in a cylindrical geometry of aspect ratio  $D/H = 2$  where  $D$  is the diameter of the cylinder. The cylindrical geometry is embedded in the Cartesian geometry using volume penalization (see Ravichandran & Wettlaufer 2020), with volume penalization parameters of  $\eta = 5 \times 10^{-3}$  or  $\eta = 10^{-3}$  giving the same results. The symmetric (SYM), ASYM and SYMFS BCs are defined in analogy with those in § 2 for the cuboidal geometry, whereas the ALLFS BC cannot be implemented with the solver used here. All other parameters,  $E = 10^{-4}$ ,  $Pr = 1$ ,  $Ra = 10^6$ , are unchanged.

In figure 8, we show the steady roll state and retrograde precessing wall modes, which should be compared with those in figure 2. The principal quantitative difference is that the wall temperature  $\theta_c^{(1)}$  up to which wall modes are sustained is larger in the cylindrical geometry for the same Rayleigh number. These wall modes travel in a retrograde or prograde direction for SYM and ASYM BCs, respectively. This is seen in the Hövmöller diagrams in figure 9, for SYMNS and ASYMFS BCs, respectively, showing the influence of asymmetric velocity BCs.

### 3.4. Tangential flow velocity at the wall

The reversal in the travel direction of the wall modes may be explained by examining the flow velocity tangential to the walls. The following arguments apply to both the cylindrical and cuboidal geometries.

For SYM velocity BCs, the time-averaged tangential velocity  $\bar{v}$  is plotted in figure 10 for insulating and fixed wall temperatures. Insulating wall BCs give a vertically symmetric tangential velocity that vanishes at the upper and lower boundaries (see also Zhang *et al.* 2020). The velocity profiles for conducting walls are starkly different.

Consider the flow at the wall with  $\theta_w = 0.9$ . Whereas for the insulating case, there is an inner,  $O(E^{-1/4})$ , Stewartson boundary layer with a prograde velocity, when the walls are at fixed temperature this boundary layer is absent and the velocities at the wall are entirely retrograde. The near-wall flow has a positive vertical velocity and turns inwards at

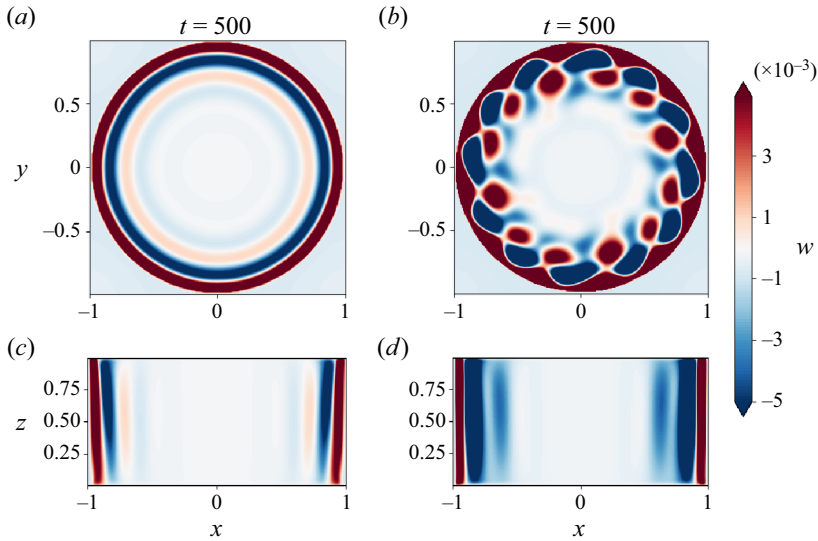


Figure 8. With  $E = 10^{-4}$ ,  $Ra = 10^6$ ,  $Pr = 1$  and SYMNS BCs in the cylindrical geometry, we see a steady convective state with wall-adjacent rolls for (a,c)  $\theta_w = 0.8$ ; and wall modes for (b,d)  $\theta_w = 0.95$ . The panels show the (a,b) horizontal and (c,d) vertical cross-sections of the vertical velocity  $w$ .

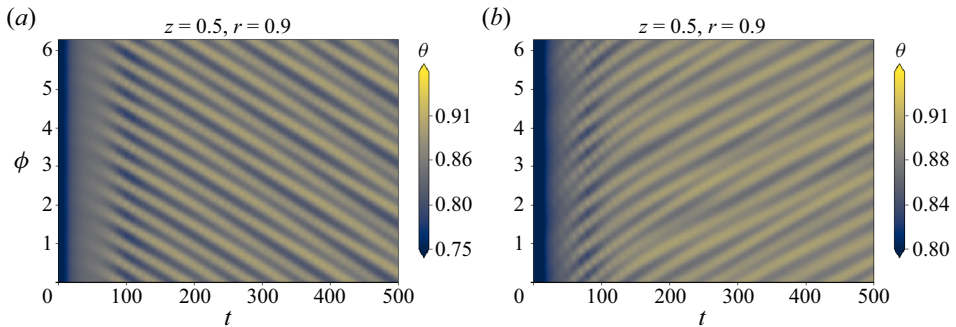


Figure 9. Hövmöller plots of the temperature  $\theta$  for  $E = 10^{-4}$ ,  $Pr = 1$ ,  $Ra = 10^6$  and  $\theta_w = 0.95$  in the cylindrical geometry shown in figure 8. The direction of propagation is (a) retrograde for SYM BCs and (b) prograde for ASYM BCs.

the upper boundary, acquiring a prograde tangential velocity. Flow towards the wall at the lower boundary acquires a retrograde tangential velocity. The stronger buoyancy-forcing at the upper boundary, due to the larger wall-normal thermal gradient, leads to a vertical shear, or thermal wind, resulting in the skewed velocity profiles seen in figure 10(a). These arguments apply when  $\theta_w = 0.1$ .

Despite the strong vertical shear seen in figure 10(a), the depth-averaged tangential velocity is retrograde, as shown in figure 10(b). The profiles for  $\theta_w = 0.1$  and  $\theta_w = 0.9$ , expected to be similar by symmetry, have the same sense as in the case with insulating BCs. In the rotation-dominated flows considered here, wall modes remain vertically coherent, as clearly seen in the three-dimensional contours of figure 11 for both SYM and ASYM BCs, and their precession direction is determined by the depth-averaged tangential velocity, as shown in figure 10(b). Thus, the precession direction of the wall modes is the same for  $\theta_w = 0.1$  and  $0.9$ , and insulating BCs.

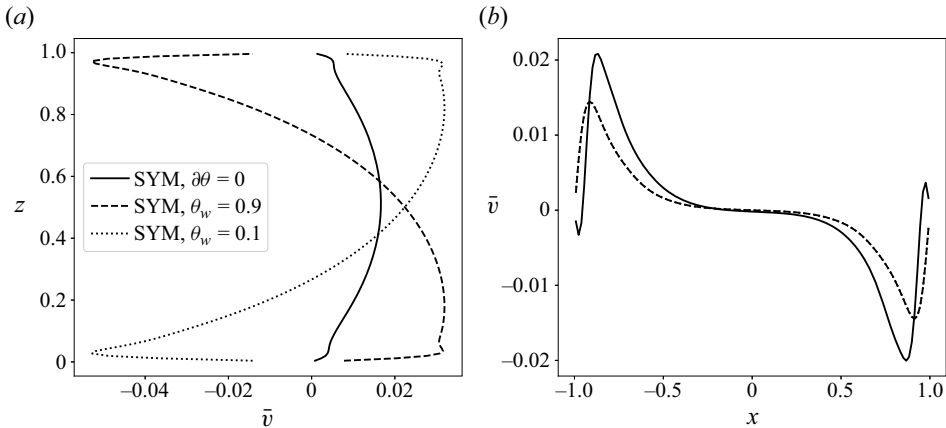


Figure 10. With  $E = 10^{-4}$ ,  $Ra = 10^6$ ,  $Pr = 1$  and SYM velocity BCs, (a) the time-averaged velocity  $\bar{v}$  tangential to the wall at  $x = -1$ . The velocities are averaged over the region  $x < -0.9$ ,  $-0.75 < y < 1.25$  and over at least 300 flow time units once the wall modes have set in. Large negative (thus prograde) velocities are seen near the upper and lower boundaries for  $\theta_w = 0.9$  and  $\theta_w = 0.1$ , respectively; (b) The tangential velocity  $\bar{v}(x)$  averaged over the region  $-0.75 < y < 1.25$ ,  $0 < z < 1$ . Despite the stark asymmetry in the vertical profiles of velocity for  $\theta_w = 0.1$  and  $\theta_w = 0.9$ , the depth-averaged velocity profiles are identical. The positive, and thus retrograde, average tangential velocities are consistent with the retrograde precession of the wall modes for these velocity BCs.

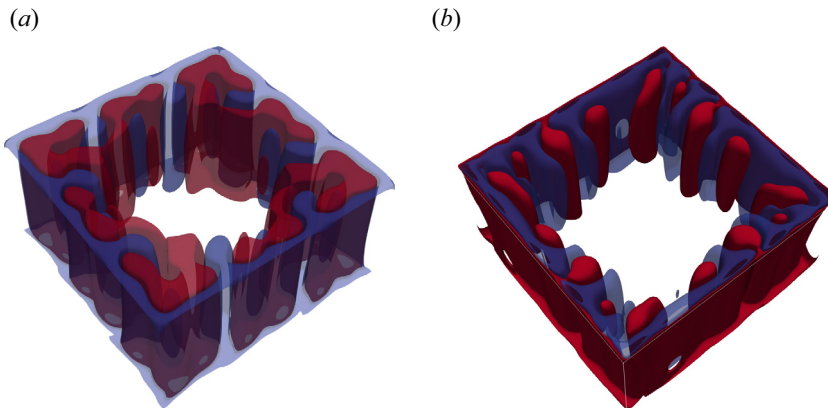


Figure 11. Three-dimensional isocontours of the vertical velocity  $w$  for  $E = 10^{-4}$ ,  $Pr = 1$ ,  $Ra = 10^6$  and (a)  $\theta_w = 0.05$  with SYMNS BCs, and (b)  $\theta_w = 0.975$  with ASYM BCs. In both cases, it is evident that the flow structures are columnar, and hence rotation-dominated. The isocontours are plotted for  $w = 0.001$  (red) and  $w = -0.001$  (blue).

The effects of ASYM BCs are shown in figure 12(a,b), where the vertical profiles for  $\theta_w = 0$  and  $\theta_w = 1$  are no longer symmetric about  $z = 0.5$ . Owing to the horizontal thermal gradient, the geostrophically balanced flow develops a vertical shear, where the sign of the shear depends on the direction of the thermal gradient. As a result, the depth-averaged velocity profile, which determines the direction of precession, is retrograde for  $\theta_w = 0$  and prograde for  $\theta_w = 1$ .

Figures 13(a) and 13(b) show that for SYM BCs the depth-averaged tangential velocity is similar for  $\theta_w = 0.25$  (steady rolls) and  $\theta_w \leq 0.1$  (wall modes). The onset of wall modes for  $\theta_w \leq 0.1$  is controlled by the vertical shear, which increases with decreasing  $\theta_w$ , rather than by the depth-averaged velocity. For ASYM BCs, the average tangential velocity is prograde, and the vertical shear increases as  $\theta_w \rightarrow 1$ . This argument is bolstered by

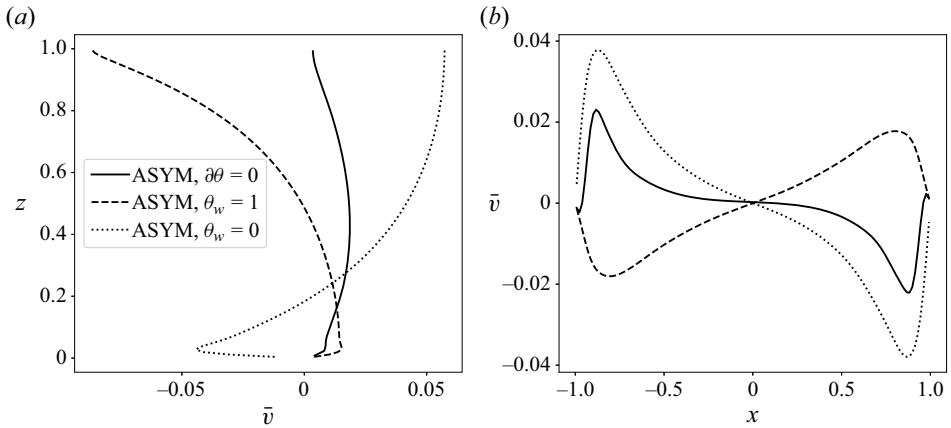


Figure 12. As in figure 10, but with ASYM velocity BCs, showing the tangential velocity  $\bar{v}$  as a function of (a)  $z$  and (b)  $x$ .

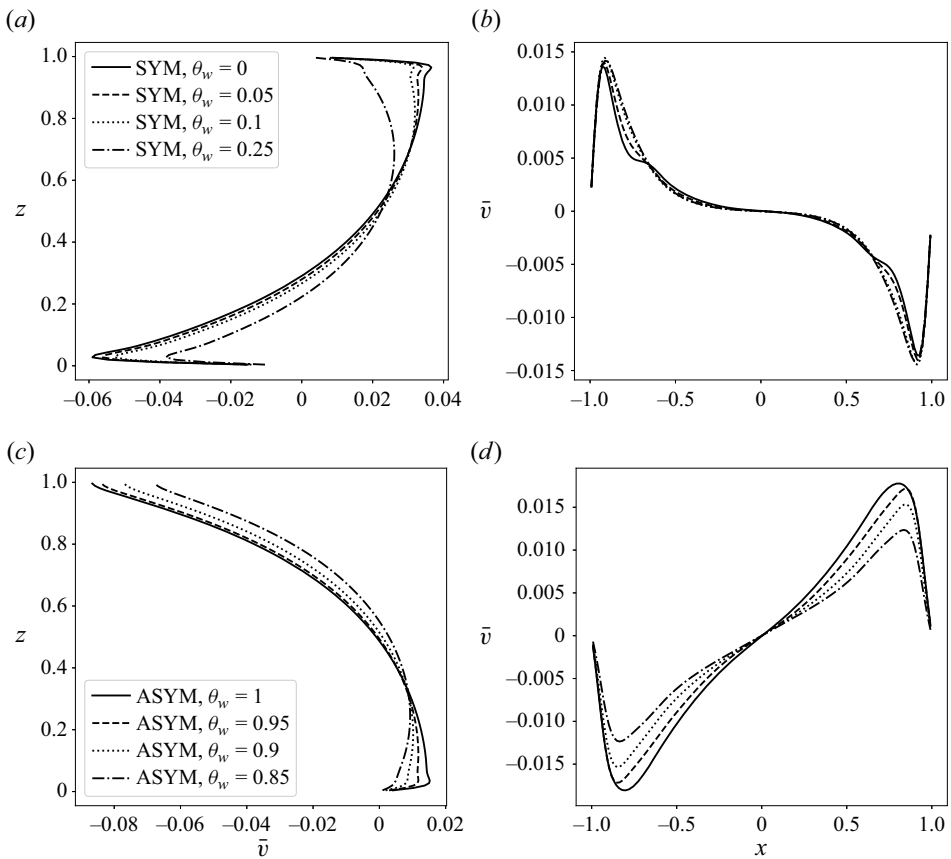


Figure 13. The tangential velocity as a function of (a,c)  $z$  and (b,d)  $x$  for cases where the flow takes the form of rolls and wall modes for (a,b) SYM and (c,d) ASYM BCs with fixed wall temperatures  $\theta_w$ . The combination of the depth-averaged tangential velocity and the vertical shear dictate the onset of wall modes from the steady roll state. Flow parameters are as in figures 10 and 12.

Wall modes in rotating Rayleigh–Bénard convection

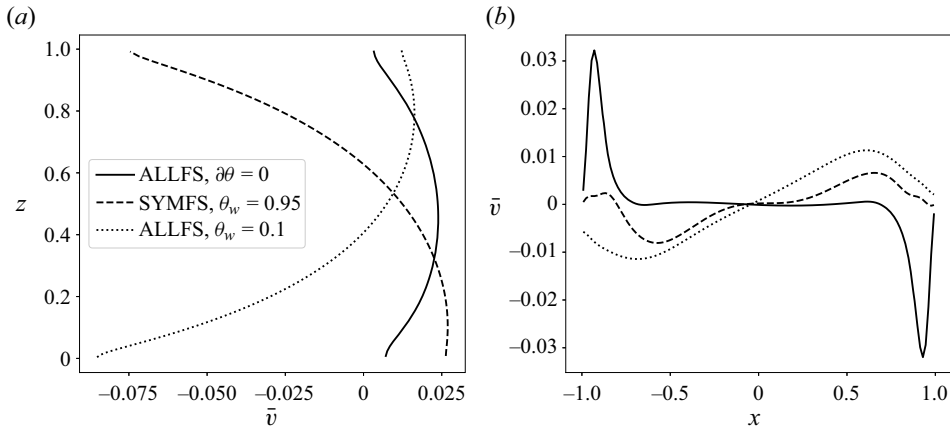


Figure 14. As in figures 10 and 12, but with SYMFS and ALLFS velocity BCs. Despite the large tangential velocities at the upper and lower boundaries seen in (a), the depth-averaged velocities in (b) are of much smaller magnitudes.

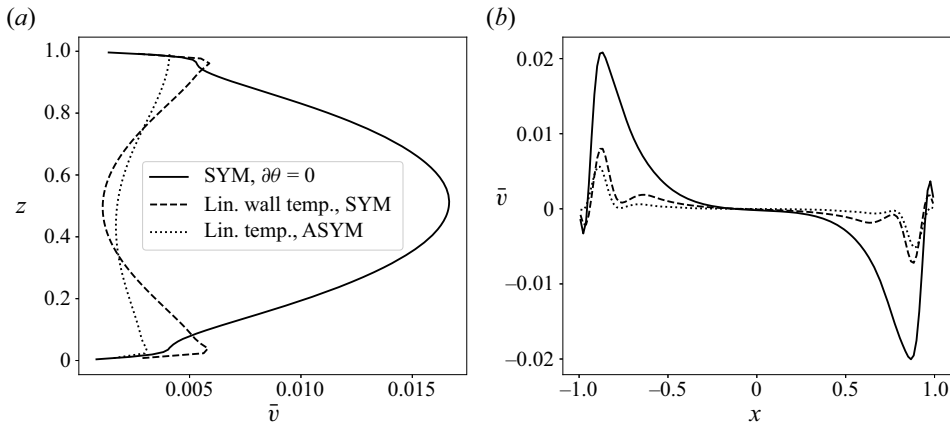


Figure 15. Tangential velocity profiles versus (a)  $z$  and (b)  $x$ , for the cases with the linear wall temperature of (1.5).

an examination of the case with the linear wall temperature profile (see (1.5)), wherein the wall thermal forcing is minimal (see Ning & Ecke 1993), and thus so too is the vertical shear. Therefore, the net tangential velocity is retrograde and the wall modes propagate in the retrograde direction for both SYM and ASYM velocity BCs. These arguments apply in both cuboidal and cylindrical geometries.

For ALLFS and SYMFS BCs, wall modes have no preferred direction of travel. In figure 14 we see that, compared with the case with insulating walls, fixed-temperature walls lead to much smaller depth-averaged tangential velocities of indeterminate direction. The latter point is demonstrated by comparing the two different realizations of the ALLFS BCs.

Lastly, figure 15 shows the tangential velocity profiles obtained with the linear wall temperature (see (1.5)), no-slip BCs on the lower boundary at  $z = 0$  and either no-slip or stress-free BCs on the upper boundary at  $z = 1$ . Since the thermal forcing at the wall is smaller than for fixed  $\theta_w$ , the vertical shear generated is negligible, and the average tangential velocity is retrograde. The wall modes propagate in a retrograde direction for both velocity BCs.

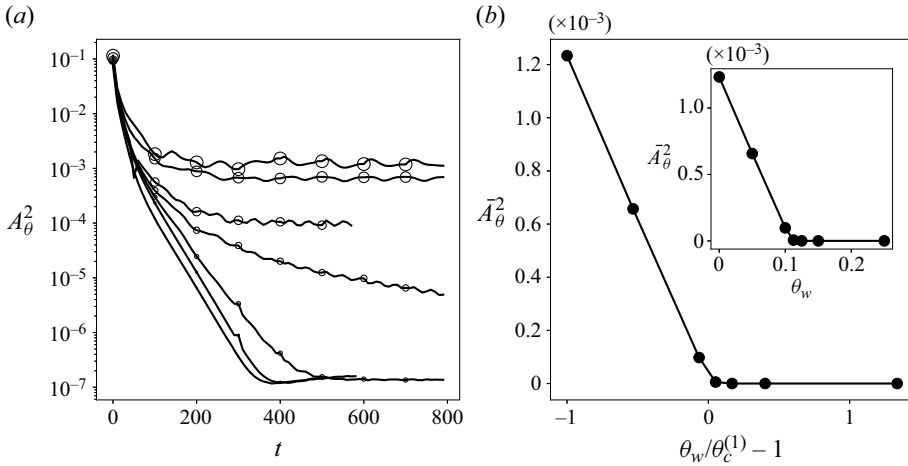


Figure 16. (a) The amplitude  $A_\theta^2$  of the wall modes as a function of time, showing the existence of a steady oscillatory state and (b) the domain-averaged oscillation amplitude  $\bar{A}_\theta^2$  as a function of the reduced wall temperature  $\theta_w/\theta_c^{(1)} - 1$  for SYMNS BCs (see table 1) and fixed flow parameters  $E = 10^{-4}$ ,  $Ra = 10^6$  and  $Pr = 1$ . In the inset of (b),  $A_\theta^2$  is plotted versus the wall temperature  $\theta_w$ . The curves in (a) are plotted for the wall temperatures  $\theta_w = (0, 0.05, 0.1, 0.1125, 0.125, 0.15, 0.25)$ , with circles of larger size for smaller  $\theta_w$ . The averages in (b) are obtained over the last 100 flow units of each curve in (a). The linear dependence of the (squared) amplitude on the magnitude of the deviation from the critical wall temperature is characteristic of a Hopf bifurcation.

### 3.5. The influence of $\theta_w$ on wall mode formation

From the results presented in §§ 3.1 and 3.2, we see that wall modes only occur for values of  $\theta_w$  sufficiently close to the upper or lower boundary temperatures, with the flow taking the form of steady rolls for  $\theta_w > \theta_c^{(1)}$  (see figure 2a,c). To determine the wall temperature  $\theta_c^{(1)}$  below which wall modes exist,  $\theta_w < \theta_c^{(1)}$ , we quantify the wall mode strength using the oscillation amplitude of the temperature  $\theta(x, y, z, t)$  as

$$A_\theta^2 = \langle (\theta - \bar{\theta})^2 \rangle, \tag{3.1}$$

where  $\bar{\theta}(x, y, z)$  is the time-averaged temperature at a given location  $(x, y, z)$ , and the angle brackets  $\langle \cdot \rangle$  denote an average over the spatial domain. Thus, the amplitude  $A_\theta = 0$  for steady (or zero) flow, and the wall temperature  $\theta_w$  at which  $A_\theta > 0$  is the critical wall temperature  $\theta_c^{(1)}$ . Similarly, the temperature  $\theta_c^{(2)}$  is defined such that when  $\theta_w > \theta_c^{(2)}$  wall modes occur, and when  $\theta_w < \theta_c^{(2)}$ , steady convective rolls occur.

In figure 16, we plot the oscillation amplitude  $A_\theta$  as the wall temperature  $\theta_w$  is varied for SYMNS BCs and fixed flow parameters  $E = 10^{-4}$ ,  $Ra = 10^6$ , and  $Pr = 1$ . We find that the domain-averaged oscillation amplitude  $\bar{A}_\theta^2 \propto |\theta_w - \theta_c^{(1)}|$ , with a critical wall temperature  $\theta_c^{(1)} \approx 0.1$ . This dependence of the amplitude on the deviation from threshold is similar to the Hopf bifurcation that occurs for wall modes with insulating walls (Ecke *et al.* 1992), in which the controlling parameter is the Rayleigh number.

We repeat this exercise for different values of  $Ra$ , keeping the parameters  $E = 10^{-4}$  and  $Pr = 1$  fixed. The smallest value of  $Ra$  such that  $\theta_c^{(1)}(Ra) > 0$  is defined as  $Ra_c^{(1)} = Ra_c^{(1)}(E, Pr, BCs)$ . The resulting behaviour for the cases we have simulated is summarized schematically in figure 17, where we find a monotonic dependence of  $\theta_c^{(1)}$  on  $Ra$ .



## Wall modes in rotating Rayleigh–Bénard convection

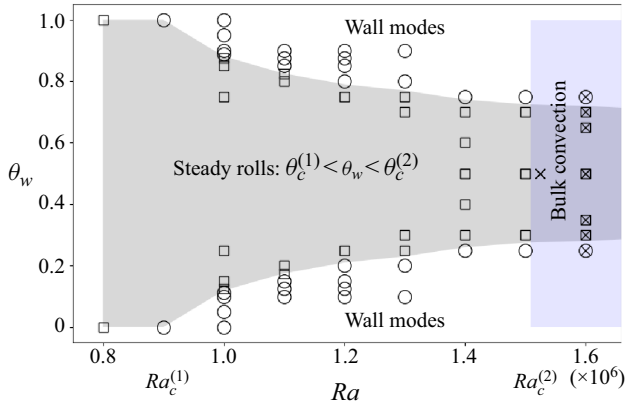


Figure 17. The different regimes of behaviour observed with conducting walls and SYMNS BCs. The symbols denote whether wall modes (circles), steady rolls (squares) or bulk convection with columnar vortices (crosses) are seen. Note that combinations of these are possible. The regions shaded grey and blue correspond to the steady roll- and bulk convective states. The lower and upper boundaries of the grey region correspond to the critical temperatures  $\theta_c^{(1)}$  and  $\theta_c^{(2)}$ , respectively. For small Rayleigh numbers  $Ra$ , wall-attached steady rolls are seen for all  $\theta_w$ . For Rayleigh numbers greater than a critical value  $Ra_c^{(1)}$ , wall modes are seen for  $\theta_w < \theta_c^{(1)}$  and  $\theta_w > \theta_c^{(2)}$ , while the steady roll state persists for  $\theta_c^{(1)} < \theta_w < \theta_c^{(2)}$ . For larger Rayleigh numbers, convection also sets in away from the walls, with the flow taking the form of a series of nested rolls that remain stable for extended periods of time. For  $Ra > Ra_c^{(2)} \equiv Ra_c^{bulk}$  of (1.2), flow in the bulk takes the form of horizontally drifting columnar vortices that are typical of rotating Rayleigh–Bénard convection, while either nested rolls or wall modes are seen adjacent to the walls. For  $E = 10^{-4}$ ,  $Pr = 1$ , we find  $Ra_c^{(1)} = 9 \times 10^5$ , and  $Ra_c^{(2)} = 1.5 \times 10^6$ .

For SYMNS BCs and wall temperatures  $\theta_w < \theta_c^{(1)}$  and  $\theta_w > \theta_c^{(2)}$  retrograde wall modes are observed as seen in figures 3(a) and 3(b), respectively. The difference  $\theta_c^{(2)} - \theta_c^{(1)}$  decreases monotonically as  $Ra$  increases.

As the Rayleigh number approaches the critical value for the onset of bulk flow,  $Ra_c^{bulk} = 1.5 \times 10^6$  from (1.2), the difference  $\theta_c^{(2)} - \theta_c^{(1)}$  can either (i) vanish before bulk flow sets in, such that  $Ra_c^{(2)} < Ra_c^{bulk}$ , or (ii) remain finite until bulk convection sets in at  $Ra_c^{(2)} = Ra_c^{bulk}$ . We find the latter case, wherein as  $Ra$  increases the flow takes the form of a series of nested convection rolls that span the entire horizontal area of the domain, as shown in figure 18(a,d). For  $Ra > Ra_c^{bulk}$ , as noted above, these rolls break down into individual columnar vortices in the fluid bulk, as seen in figure 18(b,e), due to the mechanism reported by Boubnov & Golitsyn (1986), Zhong *et al.* (2010) and Ravichandran & Wettlaufer (2020), into a state of geostrophic convection in the bulk. For larger  $Ra$ , the wall modes become less prominent.

The symmetry between  $\theta_c^{(1)}$  and  $\theta_c^{(2)}$  observed for SYMNS BCs, where  $\theta_c^{(1)} + \theta_c^{(2)} = 1$ , breaks down if the velocity boundary conditions are asymmetric, such as for ASYM BCs. In figure 19, we plot the oscillation amplitude  $A_\theta^2(\theta_w)$  for the same parameters as in figure 16, but with ASYM BCs, showing that  $\theta_c^{(1)} + \theta_c^{(2)} > 1$ .

Of further relevance to figure 17 is the fact that in the limit  $E \rightarrow 0$ , (1.3) and (1.4) for the critical Rayleigh numbers are independent of the Prandtl number. For finite  $E$ , Herrmann & Busse (1993) found that as the Prandtl number decreased so too did  $Ra_c$  (see their figures 3 and 6), driving the system towards instability. Similarly, decreasing the Ekman number increases the critical Rayleigh number, and leads to the same qualitative effects as does decreasing  $Ra$ . Next, we consider further the effects of varying the Prandtl number.

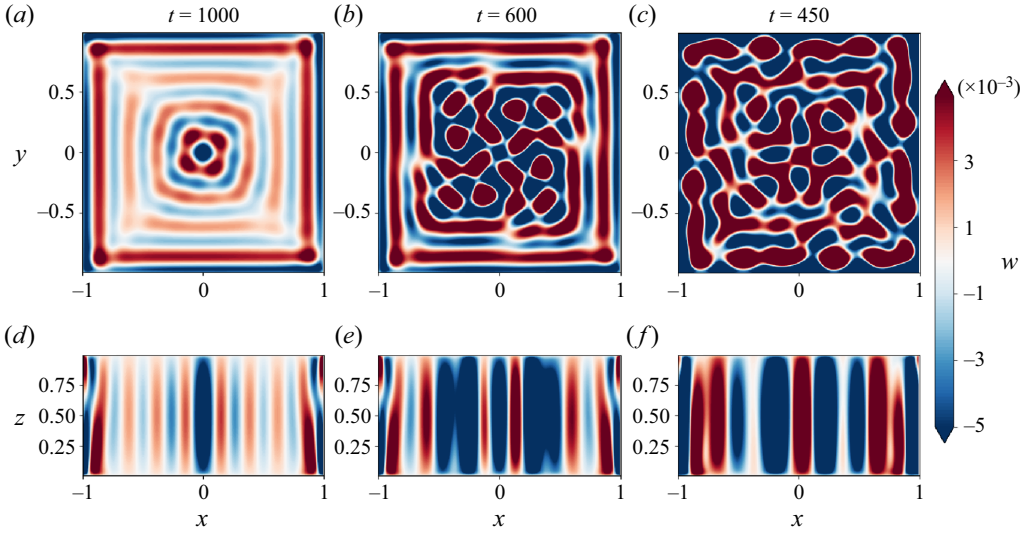


Figure 18. With  $E = 10^{-4}$ ,  $Pr = 1$ ,  $\theta_w = 0.4$  and SYMNS BCs, we see (a,d) for  $Ra = Ra_c^{bulk} = 1.5 \times 10^6$ , steady nested rolls spanning the entire horizontal extent of the domain; (b,e) wall-adjacent steady rolls with columnar vortices in the bulk for  $Ra = 1.6 \times 10^6$ . Note that for  $\theta_w = 0.25 < \theta_c^{(1)}$ , we see (c,f) bulk convection coexist with retrograde propagating wall modes. See also figure 17.

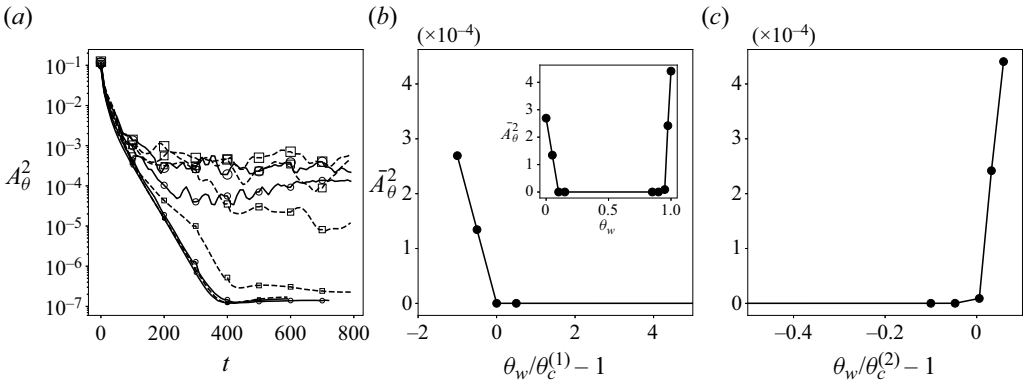


Figure 19. (a) The amplitude  $A_\theta^2$  of the spatiotemporal oscillations as a function of time, showing the existence of a steady oscillatory state, and the domain-averaged oscillation amplitude  $\bar{A}_\theta^2$  as a function of (b) the reduced wall temperature  $\theta_w/\theta_c^{(1)} - 1$ , and (c)  $\theta_w/\theta_c^{(2)} - 1$ , for ASYM BCs (see table 1). In the inset of (b),  $A_\theta^2$  is plotted versus the wall temperature  $\theta_w$ . The curves in (a) are plotted for the wall temperatures  $\theta_w = (0, 0.05, 0.1, 0.15)$  (solid lines, circles of decreasing size) and  $\theta_w = (0.85, 0.9, 0.95, 0.975, 1.0)$  (dashed lines, squares of increasing size). The averages in (b) are obtained over the last 100 flow units for each curve in (a). The linear dependence of the (squared) amplitude on the magnitude of the deviation from the critical wall temperature is characteristic of a Hopf bifurcation. Note the asymmetry between  $\theta_c^{(1)}$  and  $\theta_c^{(2)}$  (c.f. figure 16).

### 3.6. The role of the Prandtl number

An important effect of varying the Prandtl number to control the thickness of the thermal boundary layers at the walls and the resultant heat transfer. Thus, for a Boussinesq fluid, decreasing the Prandtl number is associated with increasing the thermal diffusivity.

Considering again the case of  $\theta_w \approx 1$ , with other parameters held constant, smaller Prandtl numbers result in larger buoyancy forcing at the walls, leading to larger wall-adjacent vertical velocities, and larger retrograde velocities as the flow turns inwards

## Wall modes in rotating Rayleigh–Bénard convection

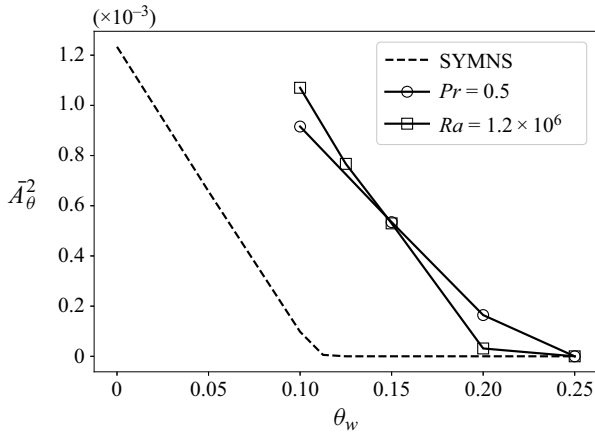


Figure 20. Oscillation amplitudes with  $E = 10^{-4}$ , as in figure 16(b), but with  $Ra = 10^6$  and  $Pr = 0.5$  (open circles), and  $Ra = 1.2 \times 10^6$ ,  $Pr = 1$  (open squares). The dashed curve is the case  $E = 10^{-4}$ ,  $Ra = 10^6$  and  $Pr = 1$  exactly as in figure 16(b).

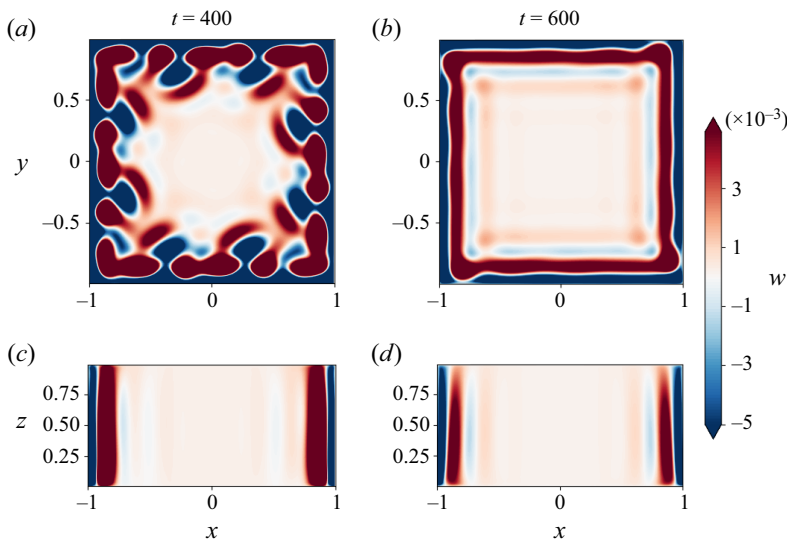


Figure 21. With  $E = 10^{-4}$ ,  $Ra = 10^6$  and SYMNS BCs, we see wall modes for (a,c)  $Pr = 0.5$  and  $\theta_w = 0.15$ ; and a steady convective state with wall-adjacent rolls for (b,d)  $Pr = 2$  and  $\theta_w = 0$ . The panels show the horizontal (a,b) and vertical (c,d) cross-sections of the vertical velocity  $w$ .

at the upper boundary. As a result, the flow experiences greater vertical shear, which results in the onset of wall modes for a smaller  $\theta_w$ . We see in figure 20 that decreasing  $Pr$  and increasing  $Ra$  both increase  $\theta_w^{(1)}$ . Thus, for  $Pr = 0.5$ , wall modes are seen for  $\theta_w = 0.15$  in figure 21(a), whereas steady rolls are seen for  $Pr = 1$ . In contrast, for  $Pr = 2$ , the steady roll state is seen for  $\theta_w = 0$  in figure 21(b), and larger values of  $Ra$  are needed for the onset of wall modes than for the onset of bulk convection. Therefore, smaller Ekman numbers, and thus more strongly rotation-dominated flows, are needed for wall modes when  $Pr > 1$ .

#### 4. Conclusion

The flow structure in confined rotating Rayleigh–Bénard convection is comprised of alternating regions of upwelling warm and downwelling cold fluid. When conditions lead to these patterns being adjacent to the walls of the system, they are commonly referred to as ‘wall-modes’, and were first observed in laboratory experiments (see Rossby (1969), Ecke *et al.* (1992), and references in the latter). Here, in geometries of aspect ratio greater than unity, we have used direct numerical simulations to study the formation and spatiotemporal evolution of wall-adjacent flow patterns when the walls are conducting instead of insulating. We showed that the wall temperature  $\theta_w$  controls whether the flow takes the form of steady convective rolls or propagating wall-modes. We found that the velocity BCs are crucial to the dynamics of wall modes, and that these modes propagate in a fixed direction only if at least one of the upper and lower boundaries obeys the no-slip condition. Moreover, the direction of propagation of the wall modes can be reversed for suitable combinations of the velocity BCs at the upper and lower boundaries and the wall temperature. In particular, if only the upper boundary is stress free, the wall modes propagate in a prograde direction when  $\theta_w \approx 1$ . Indeed, there is a similarity between the effects of velocity BCs on wall mode formation and propagation, to transient convective ring formation (Boubnov & Golitsyn 1986; Vorobieff & Ecke 1998; Zhong *et al.* 2010; Ravichandran & Wettlaufer 2020).

Although wall modes were previously shown to occur when the walls are conducting, we found that asymmetric velocity BCs drive an asymmetry in the torques exerted at the two boundaries. Such effects may have astrophysical or geophysical consequences, where asymmetric velocity BCs are common, such as for example in natural bodies of water with free upper surfaces. The effects of the asymmetry in velocity BCs on integral flow properties, such as the helicity (Moffatt & Tsinober 1992) are a subject of ongoing study.

Finally, the rich range of flow behaviour in slightly supercritical Rayleigh–Bénard convection has served as a model for the study of nonlinear-dynamical systems, such as the Benjamin–Feir and Eckhaus instabilities (Ning & Ecke 1993; Liu & Ecke 1997, 1999; Lopez *et al.* 2009). More recent studies have shown that the wall-modes in rotating Rayleigh–Bénard convection may be a topologically conserved feature, robust to severe vertically homogeneous modifications to the geometry (Favier & Knobloch 2020). Therefore, the influence of conducting walls in combination with asymmetric velocity BCs on the wall modes may provide a framework of general interest in the theory of pattern formation.

**Acknowledgements.** We thank the anonymous referees for helpful comments.

**Funding.** Computational resources from the Swedish National Infrastructure for Computing (SNIC) under grants SNIC/2020-5-471, SNIC/2021-5-449 and SNIC/2022-5-473 are gratefully acknowledged. Computations were performed on Tetralith. The Swedish Research Council, under grant no. 638-2013-9243, is gratefully acknowledged for support. Nordita is partially supported by Nordforsk. Since July 2022, S.R. has been supported through Seed Grant RD/0522-IRCCSH0-020 from IIT Bombay.

**Declaration of interests.** The authors report no conflict of interest.

**Author ORCIDs.**

© S. Ravichandran <https://orcid.org/0000-0001-9299-7570>;

© J.S. Wettlaufer <https://orcid.org/0000-0002-1676-9645>.

REFERENCES

- AURNOU, J.M., CALKINS, M.A., CHENG, J.S., JULIEN, K., KING, E.M., NIEVES, D., SODERLUND, K.M. & STELLMACH, S. 2015 Rotating convective turbulence in Earth and planetary cores. *Phys. Earth Planet. Inter.* **246**, 52–71.
- AURNOU, J.M., HORN, S. & JULIEN, K. 2020 Connections between nonrotating, slowly rotating, and rapidly rotating turbulent convection transport scalings. *Phys. Rev. Res.* **2**, 043115.
- BERCOVICI, D. 2015 Mantle dynamics, past, present and future: an overview. In *Treatise on Geophysics*, 2nd edn (ed. G. Schubert), vol. 7, chap. 1, pp. 1–42. Elsevier.
- BOUBNOV, B.M. & GOLITSYN, G.S. 1986 Experimental study of convective structures in rotating fluids. *J. Fluid Mech.* **167**, 503–531.
- CENEDESE, C. & STRANEO, F. 2023 Icebergs melting. *Annu. Rev. Fluid Mech.* **55**, 377–402.
- CHANDRASEKHAR, S. 1953 The instability of a layer of fluid heated below and subject to coriolis forces. *Proc. R. Soc. A* **217**, 306–327.
- ECKE, R.E. 2023 Rotating Rayleigh–Bénard convection: bits and pieces. *Phys. D* **444**, 133579.
- ECKE, R.E., ZHANG, X. & SHISHKINA, O. 2022 Connecting wall modes and boundary zonal flows in rotating Rayleigh–Bénard convection. *Phys. Rev. Fluids* **7**, L011501.
- ECKE, R.E., ZHONG, F. & KNOBLOCH, E. 1992 Hopf bifurcation with broken reflection symmetry in rotating Rayleigh–Bénard convection. *Europhys. Lett.* **19**, 177–182.
- EMANUEL, K. 1994 *Atmospheric Convection*. Oxford University Press.
- FAVIER, B. & KNOBLOCH, E. 2020 Robust wall states in rapidly rotating Rayleigh–Bénard convection. *J. Fluid Mech.* **895**, R1.
- GAYEN, B. & GRIFFITHS, R.W. 2022 Rotating horizontal convection. *Annu. Rev. Fluid Mech.* **54**, 105–132.
- GERBIG, K. & LAUGHLIN, G. 2022 The prospects for hurricane-like vortices in protoplanetary disks. *Astrophys. J.* **930**, 68.
- GOLDSTEIN, H.F., KNOBLOCH, E., MERCADER, I. & NET, M. 1993 Convection in a rotating cylinder. Part 1. Linear theory for moderate Prandtl numbers. *J. Fluid Mech.* **248**, 583–604.
- HERRMANN, J. & BUSSE, F.H. 1993 Asymptotic theory of wall-attached convection in a rotating fluid layer. *J. Fluid Mech.* **255**, 183–194.
- HORN, S. & SCHMID, P.J. 2017 Prograde, retrograde, and oscillatory modes in rotating Rayleigh–Bénard convection. *J. Fluid Mech.* **831**, 182–211.
- KUNNEN, R.P., CLERCX, H.J. & HEIJST, G.J.V. 2013 The structure of sidewall boundary layers in confined rotating Rayleigh–Bénard convection. *J. Fluid Mech.* **727**, 509–532.
- KUO, E.Y. & CROSS, M.C. 1993 Traveling-wave wall states in rotating Rayleigh–Bénard convection. *Phys. Rev. E* **47**, R2245.
- LIU, Y. & ECKE, R.E. 1997 Eckhaus–Benjamin–Feir instability in rotating convection. *Phys. Rev. Lett.* **78** (23), 4391–4394.
- LIU, Y. & ECKE, R.E. 1999 Nonlinear traveling waves in rotating Rayleigh–Bénard convection: stability boundaries and phase diffusion. *Phys. Rev. E* **59** (4), 4091–4105.
- LOPEZ, J.M., MARQUES, F., MERCADER, I. & BATISTE, O. 2007 Onset of convection in a moderate aspect-ratio rotating cylinder: Eckhaus–Benjamin–Feir instability. *J. Fluid Mech.* **590**, 187–208.
- LOPEZ, J.M., MARQUES, F., RUBIO, A.M. & AVILA, M. 2009 Crossflow instability of finite Bödewadt flows: transients and spiral waves. *Phys. Fluids* **21** (11), 114107.
- MARSHALL, J. & SCHOTT, F. 1999 Open-ocean convection: observations, theory, and models. *Rev. Geophys.* **37**, 1–64.
- MOFFATT, H.K. & TSINOBER, A. 1992 Helicity in laminar and turbulent flow. *Annu. Rev. Fluid Mech.* **24**, 281–312.
- NING, L. & ECKE, R.E. 1993 Rotating Rayleigh–Bénard convection: aspect-ratio dependence of the initial bifurcations. *Phys. Rev. E* **47**, 3326–3333.
- NING, L., HU, Y., ECKE, R.E. & AHLERS, G. 1993 Spatial and temporal averages in chaotic patterns. *Phys. Rev. Lett.* **71** (14), 2216.
- RAVICHANDRAN, S., TOPPALADODDI, S. & WETTLAUFER, J.S. 2022 The combined effects of buoyancy, rotation, and shear on phase boundary evolution. *J. Fluid Mech.* **941**, A39.
- RAVICHANDRAN, S. & WETTLAUFER, J.S. 2020 Transient convective spin-up dynamics. *J. Fluid Mech.* **897**, A24.
- RAVICHANDRAN, S. & WETTLAUFER, J.S. 2021 Melting driven by rotating Rayleigh–Bénard convection. *J. Fluid Mech.* **916**, A28.
- ROSSBY, H.T. 1969 A study of Bénard convection with and without rotation. *J. Fluid Mech.* **36**, 309–335.
- SCHUMACHER, J. & SREENIVASAN, K.R. 2020 Colloquium: unusual dynamics of convection in the Sun. *Rev. Mod. Phys.* **92**, 041001.

- TERRIEN, L., FAVIER, B. & KNOBLOCH, E. 2023 Suppression of wall modes in rapidly rotating Rayleigh–Bénard convection by narrow horizontal fins. *Phys. Rev. Lett.* **130** (17), 174002.
- VERONIS, G. 1966 Motions at subcritical values of the Rayleigh number in a rotating fluid. *J. Fluid Mech.* **24**, 545–554.
- VERONIS, G. 1968 Large-amplitude Bénard convection in a rotating fluid. *J. Fluid Mech.* **31**, 113–139.
- VOROBIEFF, P. & ECKE, R.E. 1998 Transient states during spin-up of a Rayleigh–Bénard cell. *Phys. Fluids* **10**, 2525–2538.
- WEDI, M., MOTURI, V.M., FUNFSCHILLING, D. & WEISS, S. 2022 Experimental evidence for the boundary zonal flow in rotating Rayleigh–Bénard convection. *J. Fluid Mech.* **939**, A14.
- WELLS, M.G. & WETTLAUFER, J.S. 2008 Circulation in lake Vostok: a laboratory analogue study. *Geophys. Res. Lett.* **35**, L03501.
- WIT, X.M.D., GUZMÁN, A.J., MADONIA, M., CHENG, J.S., CLERCX, H.J. & KUNNEN, R.P. 2020 Turbulent rotating convection confined in a slender cylinder: the sidewall circulation. *Phys. Rev. Fluids* **5**, 1–13.
- ZHANG, X., GILS, D.P.V., HORN, S., WEDI, M., ZWIRNER, L., AHLERS, G., ECKE, R.E., WEISS, S., BODENSCHATZ, E. & SHISHKINA, O. 2020 Boundary zonal flow in rotating turbulent Rayleigh–Bénard convection. *Phys. Rev. Lett.* **124**, 84505.
- ZHONG, J.Q., PATTERSON, M.D. & WETTLAUFER, J.S. 2010 Streaks to rings to vortex grids: generic patterns in transient convective spin up of an evaporating fluid. *Phys. Rev. Lett.* **105**, 5–8.